

SUPERSONIC FLUTTER OF COMPOSITE SKEW PANELS

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Summary

Supersonic flutter of a composite skew panel is determined by the finite element method. An isoparametric 16-parameter conforming plate bending displacement model is adopted. It is demonstrated that the natural frequencies and the flutter parameter  $\lambda_{cr}$  are functions of two matrices  $[\Gamma]$  and  $[\Theta]$ . The first one  $[\Gamma]$  depends exclusively on the skew angle between the sides of the skew plate; the second one  $[\Theta]$  is dependent solely on the fibre orientation angle. The study is completed by a series of examples.

I. Introduction

Several authors have dealt with supersonic flutter study of rectangular panels made from isotropic material. Of these it is important to remember Earl H. Dowell [1] [2] [3] [4], who studied the subject both from the theoretical and experimental point of view.

According to T.Y. Yang [5], the aerodynamic forces due to supersonic flow are obtained from the exact theory for linearized two-dimensional unsteady flow where no limitations on the frequency order are imposed. The aerodynamic matrix becomes location dependent and must be formulated for each element. Mervyn D. Olson [6] [7] carried out an in-depth investigation with conforming and non conforming elements. J.G. Easley and G. Luessen [8] also studied under combined normal and shear edge forces. Furthermore S. Durvasula [9] made an approximate analysis using Lagrange equations employing double Fourier sine series in oblique coordinates to represent the deflection surface. T.Y. Yang and A.D. Han [10] and Chun Mei [11] studied panel flutter considering the effect of large deformations.

As regards oblique panels made from isotropic material very little literature is available. It is interesting to note the papers by Kariappa, Somashekar

and Shar [12] [13], and above all by G.A. Oyibo [14] [15] which dealt with orthotropic panel. A closed-form rather than a modal approach has been used in the solution of the partial differential equation in the analysis to avoid convergence problems.

II. Theoretical Formulation

In the method adopted the panel is subdivided into equal parallelogrammic plate bending elements (Fig.1).

At each node point there are four generalized displacements:

$$\begin{array}{ll}
 w & \text{deflection in z-direction} \\
 \left\{ \begin{array}{l} \frac{a}{2} \frac{\partial w}{\partial x} \\ \frac{b}{2} \frac{\partial w}{\partial y} \end{array} \right. & \text{the two oblique slopes} \\
 \frac{a}{4} \frac{\partial^2 w}{\partial y \partial x} & \text{slope continuity}
 \end{array}$$

Calculation procedures are simplified by means of the isoparametric formulation. It permits the passage from the real x y coordinate system to the so-called "natural"  $\xi \eta$  coordinate system through the topologic transformation:

$$\begin{aligned}
 \xi &= -1 + \frac{2}{a} (x + t y) \\
 \eta &= -1 + \frac{2}{b} \frac{y}{c}
 \end{aligned}
 \tag{1}$$

On the  $\xi \eta$  plane, the parallelogrammic plate element becomes a square plate element with coordinates whose modulus does not exceed unity (Fig.2).

Using this formulation the nodal displacement vector results as:

$$\left\{ u_{\omega} \right\} = \left\{ w_1, w_{\xi_1}, w_{\eta_1}, w_{\xi_1 \eta_1}, w_2, \dots, \dots, w_n, w_{\xi_n}, w_{\eta_n}, w_{\xi_n \eta_n} \right\} \quad 2)$$

The deflection  $w$  and slopes are expressed in terms of isoparametric coordinates  $\xi$  and  $\eta$ :

$$w = [f] \left\{ a \right\} \quad 3)$$

$$[f] = \left[ 1, \xi, \eta, \xi^2, \xi \eta, \eta^2, \xi^3, \xi^2 \eta, \xi \eta^2, \eta^3, \xi^3 \eta, \xi^2 \eta^2, \eta^3 \xi, \xi^2 \eta^3, \eta^3 \xi^2 \right] \quad 4)$$

$$\left\{ a \right\} = \left\{ a_0, a_1, a_2, \dots, a_{15} \right\} \quad 5)$$

where (5) is the vector of the constants  $a_i$ .

Replacing the nodal coordinates in (3) and in the expressions of its derivatives  $\partial/\partial \xi$ ,  $\partial/\partial \eta$ ,  $\partial^2/\partial \xi \partial \eta$ ,  $\left\{ u_{\omega} \right\}$  may be expressed in the form:

$$\left\{ u_{\omega} \right\} = [B] \left\{ a \right\} \quad 6)$$

hence

$$w = [f] [B]^{-1} \left\{ u_{\omega} \right\} \quad 7)$$

Matrix  $[B^{-1}]$  is listed in table 1.

For simplicity it is assumed that the air flowing above the panel is parallel to x-axis and that the effect of any air entrapped below may be neglected.

Assuming two-dimensional supersonic aerodynamic theory is sufficiently accurate, the aerodynamic forces acting on the infinitesimal element  $dxdy$  of the panel, valid for  $M > 1.7$ , are:

$$\Delta F_a = -\frac{2g}{\beta} \frac{\partial w}{\partial x} dx dy \quad 8)$$

which is the real aerodynamic force, proportional to the  $\partial w/\partial x$  slope and

$$\Delta F_d = -\left( \frac{2g}{V} \frac{M^2 - 2}{\beta^3} \frac{\partial w}{\partial t} \right) dx dy \quad 9)$$

is the aerodynamic damping due to the deflection  $w$ .  $\Delta F_d$  can be considered as an addition to the inertial force

$$\Delta F_m = -\rho \frac{\partial^2 w}{\partial t^2} dxdy \quad 10)$$

In this case, only  $\Delta F_a$  constitutes the aerodynamic force.

Furthermore, if  $N_x$  is the initial in-plane loading per unit of length (tension positive) parallel to x-axis, it will create a force in z-direction

$$\Delta F_z = -N_x \frac{\partial^2 w}{\partial x^2} dx dy \quad 11)$$

The deflection  $w$  is assumed to be an exponential temporal function

$$w(t) = w e^{\omega t} \quad 12)$$

with  $\omega = a+ib$  and  $w$  given by (3).

Therefore resorting to the isoparametric formulation:

$$dxdy = \det \mathcal{J} d\xi d\eta = \frac{ab}{4} d\xi d\eta \quad 13)$$

where

$$\mathcal{J} = \begin{bmatrix} \frac{dx}{d\xi} & \frac{dy}{d\xi} \\ \frac{dx}{d\eta} & \frac{dy}{d\eta} \end{bmatrix} = \begin{bmatrix} \frac{a}{2} & 0 \\ -\frac{bs}{2} & \frac{bc}{2} \end{bmatrix} \quad 14)$$

is the Jacobian operator.

Considering (12) and (13) the expression of the infinitesimal forces are:

$$\Delta F_m = -\det \mathcal{J} \rho \omega^2 w e^{\omega t} d\xi d\eta \quad 15)$$

$$\Delta F_a = -\det \mathcal{J} \frac{2g}{\beta} \frac{\partial \xi}{\partial x} \frac{\partial w}{\partial \xi} e^{\omega t} d\xi d\eta \quad 16)$$

$$\Delta F_d = -\det \mathcal{J} \frac{2g}{V} \frac{M^2 - 2}{\beta^3} \omega w e^{\omega t} d\xi d\eta \quad 17)$$

$$\Delta F_z = -\det \mathcal{J} N_x \left( \frac{\partial \xi}{\partial x} \right)^2 \frac{\partial^2 w}{\partial \xi^2} e^{\omega t} d\xi d\eta \quad 18)$$

## II. Flutter Equation

In order to express the forces at the nodes, the principle of virtual work is used:

$$W_e + W_a + W_m + W_n = W_i \quad (19)$$

where

- $W_e$  external work
- $W_a$  work due to aerodynamic forces
- $W_m$  work due to inertial forces and aerodynamic damping
- $W_n$  work due to  $\Delta F_n$  created by  $N_x$
- $W_i$  internal work

In [19] it is demonstrated that formula (19) governing motion can be written in the following adimensional form:

$$\left\{ \bar{F}_e \right\} = \frac{b}{4a} c \left[ \lambda \left[ \bar{A}_e \right] - \mu \left[ \bar{M}_e \right] + R_x \left[ \bar{N}_e \right] + \left[ \bar{K}_e \right] \right] \left\{ u_e \right\} \quad (20)$$

where

$$\lambda = \frac{2qL^3}{\beta D} \quad \text{and} \quad \mu = - \frac{\rho L^4 \omega^2}{D} - \lambda \frac{L(M^2-2)\omega}{\beta^2 V}$$

are parameter and eigenvalues of adimensional aerodynamic pressure respectively, and

$$R_x = \left[ L^2 / n^2 D \right] N_x$$

$$\left[ \bar{A}_e \right] = 2/n^3 \left[ A_e \right]$$

$$\left[ \bar{M}_e \right] = 1/n^4 \left[ M_e \right]$$

$$\left[ \bar{N}_e \right] = \left[ N_e \right]$$

$$\left[ \bar{K}_e \right] = \frac{16}{D} \left[ K_e \right]$$

$$\left\{ \bar{F}_e \right\} = \left\{ F_e \right\} L^2 / n^2 D$$

$\left\{ \bar{F}_e \right\}$  = generalized adimensional nodal forces corresponding to generalized displacements

Assembling finite elements and applying kinematic boundary conditions, the instability condition of the panel examined can be written in the following way:

$$\left[ \left[ \bar{K} \right] + \lambda \left[ \bar{A} \right] - \mu \left[ \bar{M} \right] + \left[ \bar{N} \right] \right] \left\{ u \right\} = 0 \quad (21)$$

The eigenvalues can be put in a more convenient form:

$$\mu = -g_A \left( \frac{\omega}{\omega_0} \right) - \left( \frac{\omega}{\omega_0} \right)^2 \quad (22)$$

with

$$g_A = \frac{(M^2-2) \rho_a V}{\beta^3 \rho \omega_0} \quad (23)$$

damping aerodynamic adimensional parameter and

$$\omega_0 = \sqrt{\frac{D}{\rho a^4}} \quad (24)$$

where  $\omega_0$  is a convenient frequency scale.  $g_A$  varies approximately between 0 and 50 [17].

### Results and Concluding Remarks

Several application examples of the present method are presented in this section. In the case studied for a composite skew angle panel, bending stiffness is:

$$D = \sqrt{E_1 E_2} / 12(1 - \nu_{12} \nu_{21})$$

In particular for isotropic panel

$$E = E_1 = E_2 \quad \nu = \nu_{12} = \nu_{21}$$

The numerical matrices  $\left[ \bar{A}_e \right]$ ,  $\left[ \bar{M}_e \right]$  and  $\left[ \bar{N}_e \right]$  are calculated and are listed in table 2.

The adimensional stiffness matrix  $\bar{K}_e$  has been calculated for symmetric laminates with multiple orthotropic layers. All elements of  $\left[ K_e \right]$  are functions of six terms (Ref. Table 2)

#### IV. Symbols

$$\begin{Bmatrix} \bar{k}_{4,4} \\ \frac{b}{a} \bar{k}_{4,5} \\ \frac{b^2}{a^2} \bar{k}_{4,6} \\ \frac{b^2}{a^2} \bar{k}_{5,5} \\ \frac{b^3}{a^3} \bar{k}_{5,6} \\ \frac{b^4}{a^4} \bar{k}_{6,6} \end{Bmatrix} = \frac{1024}{h^3 \sqrt{E_1 E_2}} [\Gamma] \sum_{k=1}^N (z_k^3 - z_{k-1}^3) [\Theta]_k \begin{Bmatrix} E_1 \\ E_2 \nu_{12} \\ E_2 \\ G \end{Bmatrix}$$

that depend on the angles  $\gamma$  and  $\vartheta$  by means of the matrices

$$[\Gamma] = \begin{bmatrix} 1 & 2t^2 & t^4 & 4t & 4t^3 & 4t^2 \\ 0 & t/c & t^3/c & 1/c & 3t^2/c & 2t/c \\ 0 & 1/c^2 & t^2/c^2 & 0 & 2t/c^2 & 0 \\ 0 & 0 & t^2/c^2 & 0 & 2t/c^2 & 1/c^2 \\ 0 & 0 & t/c^3 & 0 & 1/c^3 & 0 \\ 0 & 0 & 1/c^4 & 0 & 0 & 0 \end{bmatrix}$$

and for the k-th layer

$$[\Theta]_k = (c \theta^4)_k \begin{bmatrix} 1 & 2t\theta^2 & t\theta^4 & 4t\theta^2 \\ t\theta^2 & t\theta^4 + 1 & t\theta^2 & -4t\theta^2 \\ t\theta^4 & 2t\theta^2 & 1 & 4t\theta^2 \\ t\theta & -t\theta(1-t\theta^2) & -t\theta^3 & -2t\theta(1-t\theta^2) \\ t\theta^3 & t\theta(1-t\theta^2) & -t\theta & 2t\theta(1-t\theta^2) \\ t\theta^2 & -2t\theta^2 & t\theta^2 & (1-t\theta^2)^2 \end{bmatrix}$$

The elements of  $[\bar{K}_e]$  are shown in table 3.

The calculation method used is first of all to set the master aerodynamic-stiffness matrix of equation (21) for a particular assemblage of finite elements.

The eigenvalues are then calculated for increasing values of the dynamic parameter  $\lambda$  until the first coalescence occurs.

The graphics in Figs. 3, 4 and 5 show the variation of  $\lambda_{cr} / \sqrt{\mu_{cr}}$  and the natural frequencies  $\sqrt{\mu}$  (for  $\lambda = 0$ ) of various composite panels simply supported and clamped on all sides as  $\gamma$  and  $\vartheta$  vary.

An eight-layered symmetric lay-up  $(\theta, -\theta, \theta, -\theta, -\theta, \theta, -\theta, \theta)$  is used and the material properties of typical unidirectional composites are taken from [16]. The first natural frequency values may be compared with [16], and all parameter  $\lambda_{cr} / \mu_{cr}$  and  $\mu$  with [19], [20].

$\gamma$	skew angle
c s t	cos $\gamma$ sin $\gamma$ tg $\gamma$
$\vartheta$	fibre orientation
$c_\vartheta$ $t_\vartheta$	cos $\vartheta$ tg $\vartheta$
x, y	rectangular coordinate axes
$\bar{y}$	axis along the oblique side of parallelogrammic plate element
L	length of plate along x
$n^2$	finite element grid
N	number of layers
$\xi, \eta$	natural coordinate axes corresponding to x, y
a b	lengths of sides of parallelogrammic plate element
h	plate thickness
w	transverse displacement in the z-direction
$E_1$ $E_2$	Young's moduli in the two principal directions
$\nu_{12}$	major Poisson's ratio
$\nu_{21}$	minor Poisson's ratio
$G_{12}$	shear modulus in the panel plane
$G =$	$G_{12}(1 - \nu_{12}\nu_{21})$
V	speed of air flow
$\rho$	density per surface unit of the pannel
$\rho_a$	air density
q =	$\rho_a V^2 / 2$
M	Mach number
$\beta =$	$\sqrt{M^2 - 1}$
$\lambda, \lambda_{cr}$	nondimensional dynamic pressure and coalescence value, respectively

The other symbols are defined in the paper.

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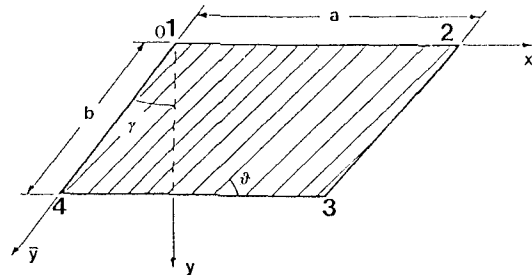


FIGURE 1

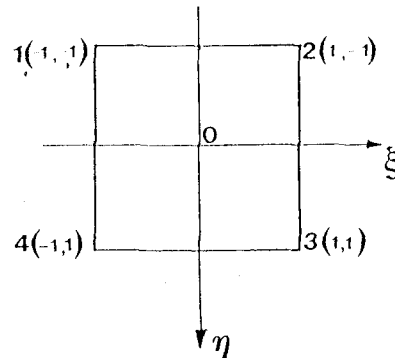


FIGURE 2

$\gamma = 15^\circ$

number of finite elements = 16

□ Boron/Epoxy

◇ Glass/Epoxy

+ Graphite/Epoxy

△ Kevlar/Epoxy

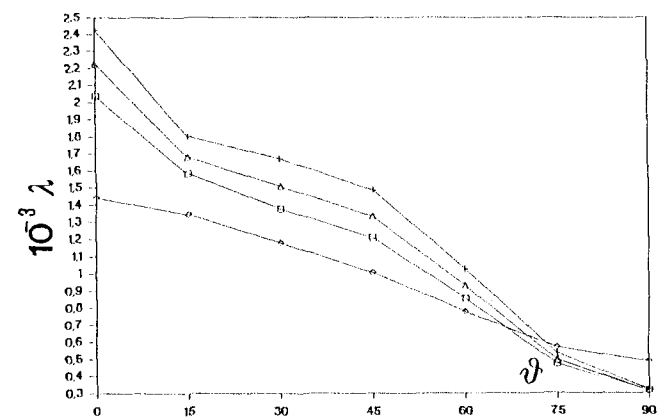
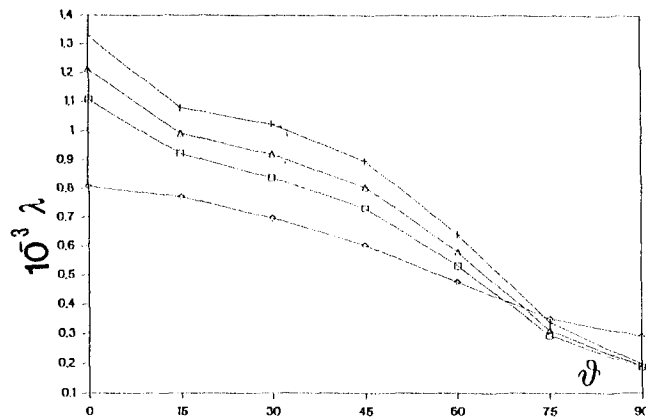
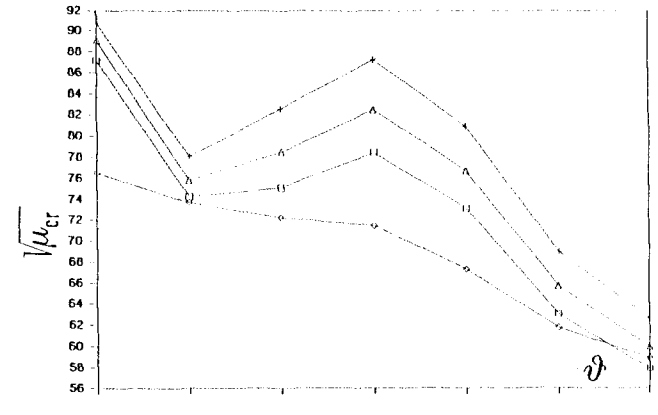
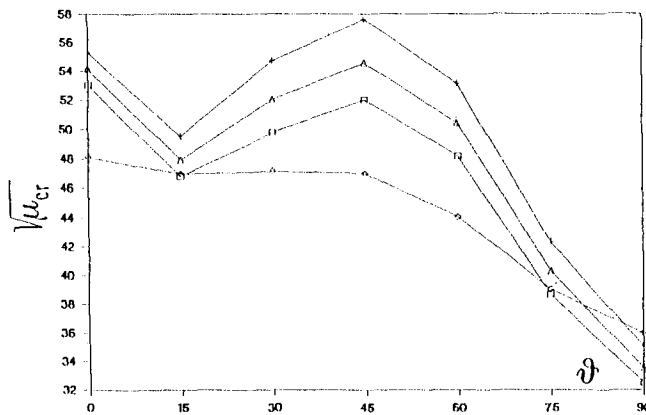
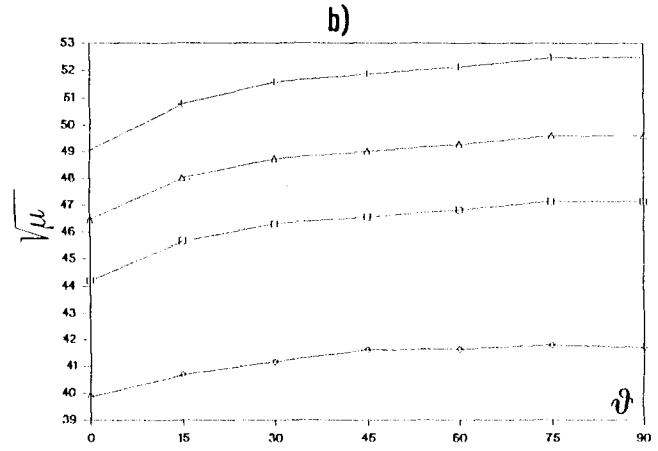
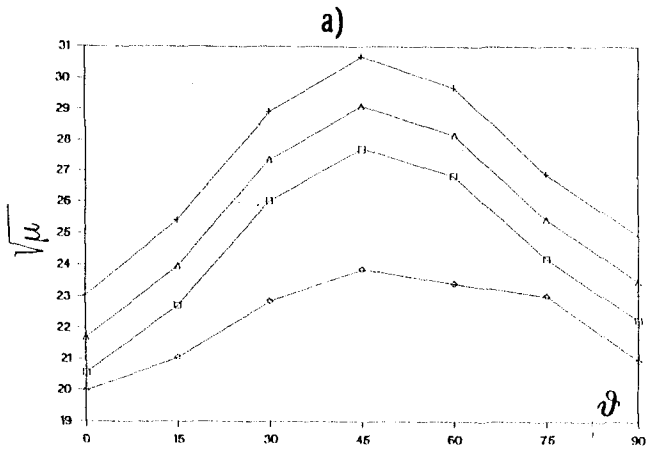


FIGURE 3 Simply supported rhombic panel (a)  
Clamped panel (b)

$\gamma=30^\circ$

number of finite elements = 16

□ Boron/Epoxy

◇ Glass/Epoxy

+ Graphite/Epoxy

△ Kevlar/Epoxy

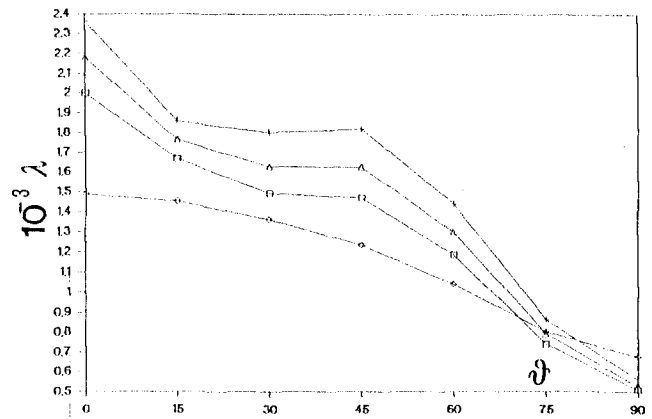
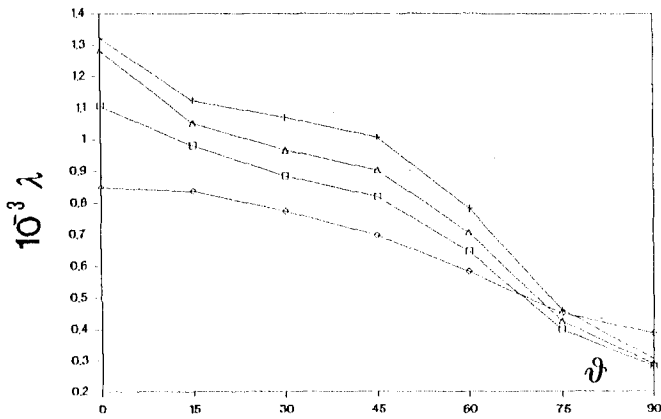
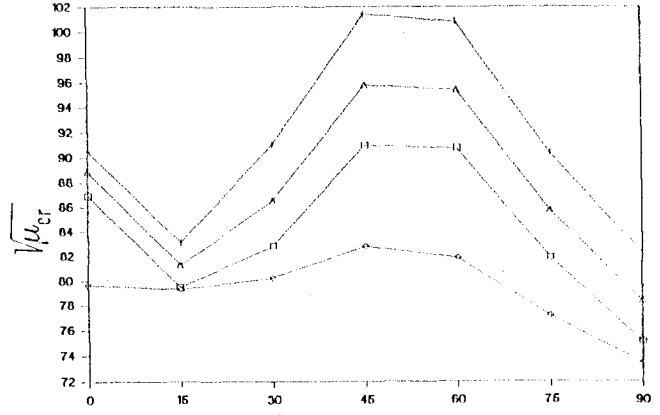
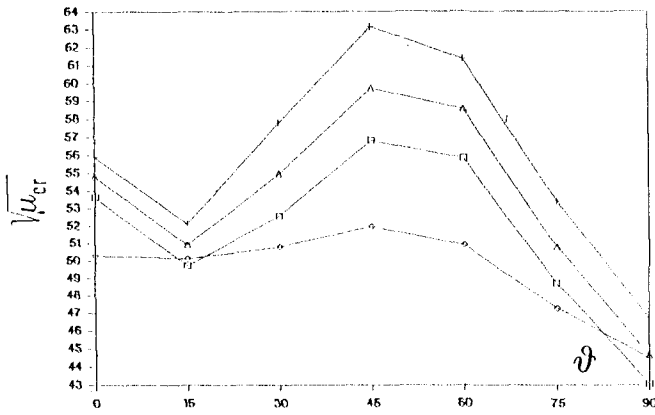
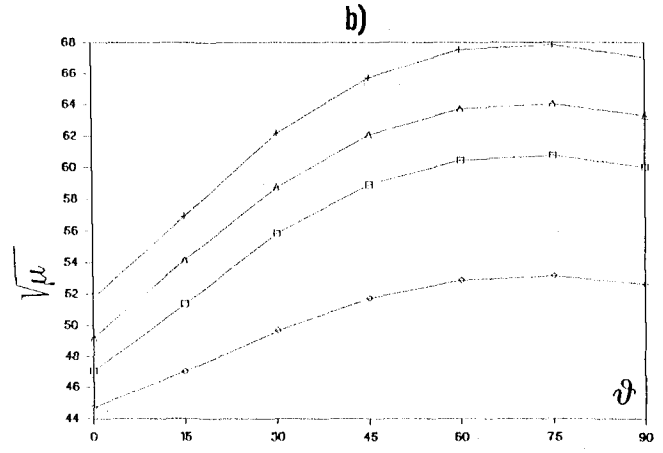
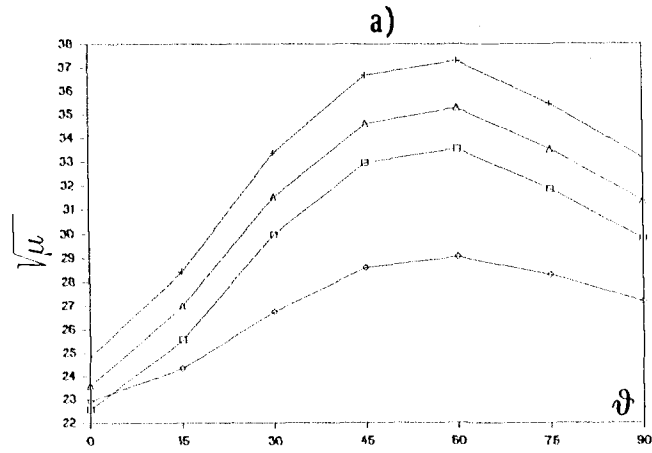


FIGURE 4 Simply supported rhombic panel (a)

Clamped panel (b)

$\gamma=45^0$

number of finite elements = 16

□ Boron/Epoxy

◇ Glass/Epoxy

+ Graphite/Epoxy

△ Kevlar/Epoxy

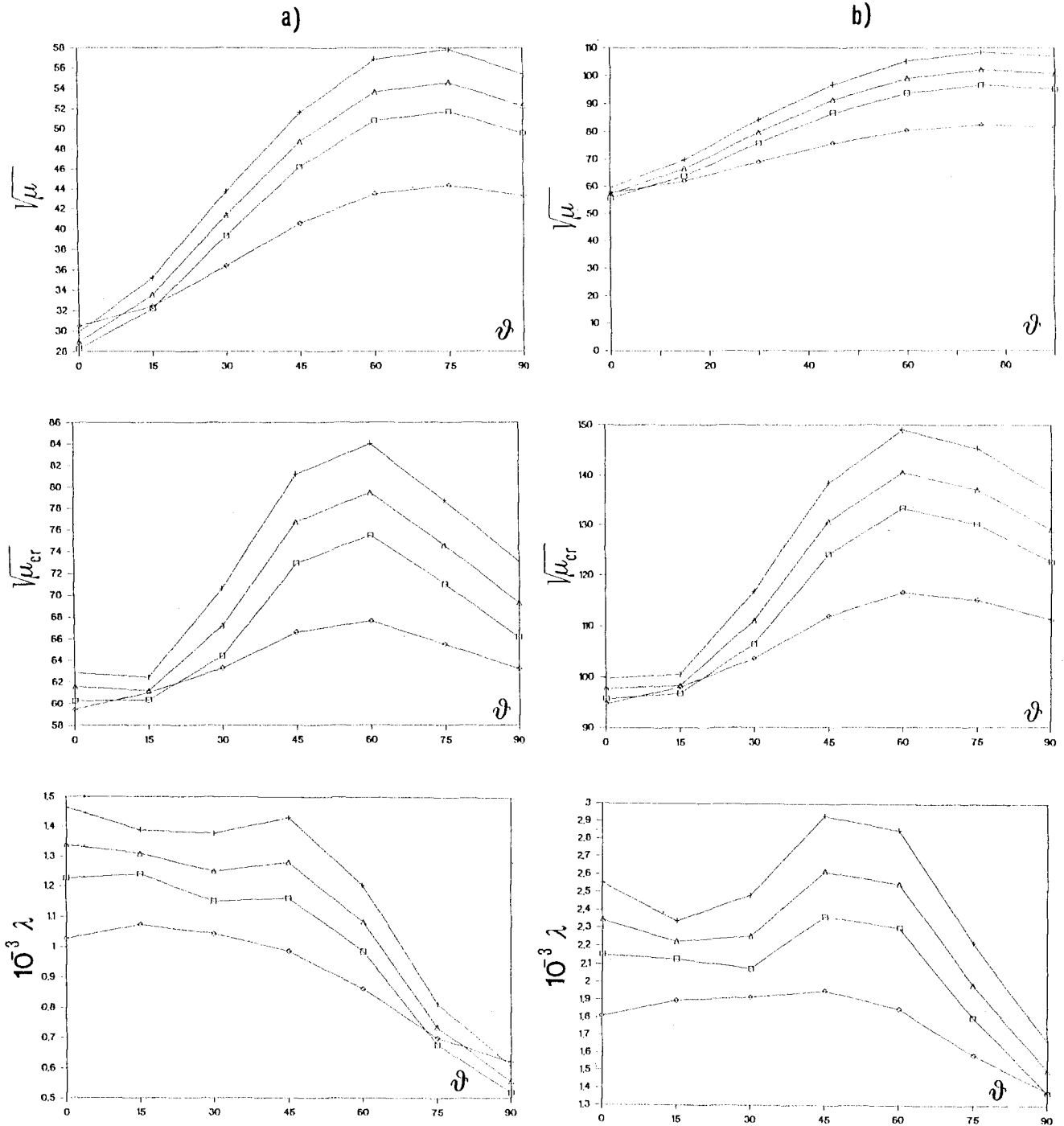


FIGURE 5 Simply supported rhombic panel (a)

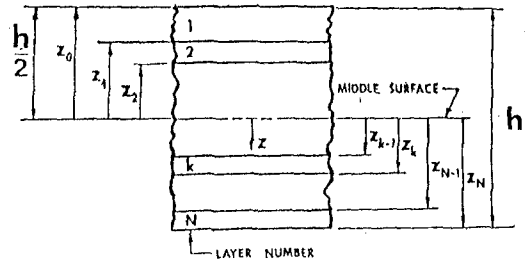
Clamped panel (b)



TABLE 1

$$[B]^{-1} = \frac{1}{16} \begin{bmatrix} 4 & 2 & 2 & 1 & 4 & -2 & 2 & -1 & 4 & -2 & -2 & 1 & 4 & 2 & -2 & -1 \\ -6 & -2 & -3 & -1 & 6 & -2 & 3 & -1 & 6 & -2 & -3 & -1 & -6 & -2 & 3 & 1 \\ -5 & -3 & -2 & -1 & -6 & 3 & -2 & 1 & 6 & -3 & -2 & 1 & 6 & 3 & -2 & -1 \\ 0 & -2 & 0 & -1 & 0 & 2 & 0 & 1 & 0 & 2 & 0 & -1 & 0 & -2 & 0 & 1 \\ 9 & 3 & 3 & 1 & -9 & 3 & -3 & 1 & 9 & -3 & -3 & 1 & -9 & -3 & 3 & 1 \\ 0 & 0 & -2 & -1 & 0 & 0 & -2 & 1 & 0 & 0 & 2 & -1 & 0 & 0 & 2 & 1 \\ 2 & 2 & 1 & 1 & -2 & 2 & -1 & 1 & -2 & 2 & 1 & -1 & 2 & 2 & -1 & -1 \\ 0 & 3 & 0 & 1 & 0 & -3 & 0 & -1 & 0 & 3 & 0 & -1 & 0 & -3 & 0 & 1 \\ 0 & 0 & 3 & 1 & 0 & 0 & -3 & 1 & 0 & 0 & 3 & -1 & 0 & 0 & -3 & -1 \\ 2 & 1 & 2 & 1 & 2 & -1 & 2 & -1 & -2 & 1 & 2 & -1 & -2 & -1 & 2 & 1 \\ -3 & -3 & -1 & -1 & 3 & -3 & 1 & -1 & -3 & 3 & 1 & -1 & 3 & 3 & -1 & -1 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & -1 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & -1 \\ -3 & -1 & -3 & -1 & 3 & -1 & 3 & -1 & -3 & 1 & 3 & -1 & 3 & 1 & -3 & -1 \\ 0 & 0 & -1 & -1 & 0 & 0 & 1 & -1 & 0 & 0 & -1 & 1 & 0 & 0 & 1 & 1 \\ 0 & -1 & 0 & -1 & 0 & 1 & 0 & 1 & 0 & -1 & 0 & 1 & 0 & 1 & 0 & -1 \\ 1 & 1 & 1 & 1 & -1 & 1 & -1 & 1 & -1 & -1 & 1 & -1 & -1 & 1 & 1 & 1 \end{bmatrix}$$

TABLE 2



Geometry of an N-layered laminate.

$\bar{k}_{4,11} = \bar{k}_{4,5}$	$\bar{k}_{7,7} = 3\bar{k}_{4,4}$	$\bar{k}_{9,9} = (\bar{k}_{9,6} + 4\bar{k}_{5,5})/3$
$\bar{k}_{4,13} = \bar{k}_{4,5}$	$\bar{k}_{7,9} = 2\bar{k}_{4,5}$	$\bar{k}_{9,10} = 2\bar{k}_{5,6}$
$\bar{k}_{5,11} = \bar{k}_{5,5}$	$\bar{k}_{7,9} = \bar{k}_{4,6}$	$\bar{k}_{10,10} = 3\bar{k}_{6,6}$
$\bar{k}_{5,13} = \bar{k}_{5,5}$	$\bar{k}_{9,9} = (\bar{k}_{4,4} + 4\bar{k}_{4,5})/3$	$\bar{k}_{11,11} = (5\bar{k}_{4,4} + 9\bar{k}_{5,5})/5$
$\bar{k}_{6,11} = \bar{k}_{5,6}$	$\bar{k}_{9,9} = 2(\bar{k}_{4,5} + \bar{k}_{5,6})/3$	$\bar{k}_{11,13} = \bar{k}_{4,6} + \bar{k}_{5,5}$
$\bar{k}_{6,13} = \bar{k}_{5,6}$	$\bar{k}_{9,10} = \bar{k}_{4,6}$	$\bar{k}_{13,13} = (5\bar{k}_{6,6} + 9\bar{k}_{5,5})/5$
$\bar{k}_{4,12} = 1/3(\bar{k}_{4,6} + \bar{k}_{4,4})$		$\bar{k}_{9,15} = \bar{k}_{4,4}/5 + (4\bar{k}_{5,5} + \bar{k}_{4,6})/3$
$\bar{k}_{5,12} = 1/3(\bar{k}_{5,6} + \bar{k}_{4,5})$		$\bar{k}_{9,15} = 2\bar{k}_{4,5}/5 + 4\bar{k}_{5,6}/3$
$\bar{k}_{6,12} = 1/3(\bar{k}_{6,6} + \bar{k}_{4,6})$		$\bar{k}_{10,15} = \bar{k}_{6,6} + 3\bar{k}_{4,6}/5$
$\bar{k}_{11,12} = 3/5\bar{k}_{5,6} + 5/3\bar{k}_{4,5}$		$\bar{k}_{14,15} = 8(\bar{k}_{5,6} + \bar{k}_{4,5})/5$
$\bar{k}_{12,13} = 3/5\bar{k}_{4,5} + 5/3\bar{k}_{5,6}$		$\bar{k}_{4,16} = \bar{k}_{4,5}$
$\bar{k}_{12,12} = (\bar{k}_{4,4} + \bar{k}_{6,6})/5 + (16\bar{k}_{5,5} + 2\bar{k}_{4,6})/9$		$\bar{k}_{5,16} = \bar{k}_{5,5}$
$\bar{k}_{7,14} = 3/5\bar{k}_{6,6} + \bar{k}_{4,4}$		$\bar{k}_{11,16} = 3(3\bar{k}_{5,5} + \bar{k}_{4,4} + \bar{k}_{4,6})/5$
$\bar{k}_{8,14} = 2/5\bar{k}_{5,6} + 4/3\bar{k}_{4,5}$		$\bar{k}_{12,16} = 7(\bar{k}_{5,6} + \bar{k}_{4,5})/5$
$\bar{k}_{9,14} = \bar{k}_{5,6}/5 + (4\bar{k}_{5,5} + \bar{k}_{4,6})/3$		$\bar{k}_{13,16} = 3(3\bar{k}_{5,5} + \bar{k}_{6,6} + \bar{k}_{4,6})/5$
$\bar{k}_{10,14} = 2\bar{k}_{5,6}$		$\bar{k}_{14,14} = \bar{k}_{6,6}/7 + (3\bar{k}_{4,4} + 12\bar{k}_{5,5} + 2\bar{k}_{4,6})/5$
$\bar{k}_{15,15} = \bar{k}_{4,4}/7 + (3\bar{k}_{6,6} + 12\bar{k}_{5,5} + 2\bar{k}_{4,6})/5$		
$\bar{k}_{7,15} = 2\bar{k}_{4,5}$		$\bar{k}_{16,16} = 3(\bar{k}_{4,4} + \bar{k}_{6,6})/7 + 9(9\bar{k}_{5,5} + 2\bar{k}_{4,6})/25$

$$[\bar{K}_0] = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ & & \bar{k}_{4,4} & \bar{k}_{4,5} & \bar{k}_{4,6} & 0 & 0 & 0 & 0 & \bar{k}_{4,11} & \bar{k}_{4,12} & \bar{k}_{4,13} & 0 & 0 & 0 & \bar{k}_{4,16} \\ & & \bar{k}_{5,5} & \bar{k}_{5,6} & 0 & 0 & 0 & 0 & \bar{k}_{5,11} & \bar{k}_{5,12} & \bar{k}_{5,13} & 0 & 0 & 0 & 0 & \bar{k}_{5,16} \\ & & & \bar{k}_{6,6} & 0 & 0 & 0 & 0 & \bar{k}_{6,11} & \bar{k}_{6,12} & \bar{k}_{6,13} & 0 & 0 & 0 & 0 & \bar{k}_{6,16} \\ & & & & \bar{k}_{7,7} & \bar{k}_{7,9} & \bar{k}_{7,9} & 0 & 0 & 0 & 0 & \bar{k}_{7,14} & \bar{k}_{7,15} & 0 & 0 & 0 \\ & & & & & \bar{k}_{8,8} & \bar{k}_{8,9} & \bar{k}_{8,9} & \bar{k}_{8,10} & 0 & 0 & 0 & \bar{k}_{8,14} & \bar{k}_{8,15} & 0 & 0 \\ & & & & & & \bar{k}_{9,9} & \bar{k}_{9,9} & \bar{k}_{9,10} & 0 & 0 & 0 & \bar{k}_{9,14} & \bar{k}_{9,15} & 0 & 0 \\ & & & & & & & \bar{k}_{10,10} & 0 & 0 & 0 & 0 & \bar{k}_{10,14} & \bar{k}_{10,15} & 0 & 0 \\ & & & & & & & & \bar{k}_{11,11} & \bar{k}_{11,12} & \bar{k}_{11,13} & 0 & 0 & 0 & 0 & \bar{k}_{11,16} \\ & & & & & & & & & \bar{k}_{12,12} & \bar{k}_{12,13} & 0 & 0 & 0 & 0 & \bar{k}_{12,16} \\ & & & & & & & & & & \bar{k}_{13,13} & 0 & 0 & 0 & 0 & \bar{k}_{13,16} \\ & & & & & & & & & & & \bar{k}_{14,14} & \bar{k}_{14,15} & 0 & 0 & 0 \\ & & & & & & & & & & & & \bar{k}_{15,15} & 0 & 0 & 0 \\ & & & & & & & & & & & & & \bar{k}_{16,16} & 0 & 0 \end{bmatrix}$$

TABLE 3

$$[\bar{M}_e] = \frac{1}{11025}$$

6084	1716	1716	484	2106	-1014	594	-286	729	-351	-351	169	2106	594	-1014	-286
	624	484	176	1014	-468	286	-132	351	-162	-169	78	594	216	-286	-104
		624	176	594	-286	216	-104	351	-169	-162	78	1014	286	-468	-132
			64	286	-132	104	-48	169	-78	-78	36	286	104	-132	-48
				6084	-1716	1716	-484	2106	-594	-1014	286	729	351	-351	-169
					624	-484	176	-594	216	286	-104	-351	-162	169	78
						624	-176	1014	-286	-468	132	351	169	-162	-78
							64	-286	104	132	-48	-169	-78	78	36
								6084	-1716	-1716	484	2106	1014	-594	-286
									624	484	-176	-1014	-468	286	132
										624	-176	-594	-286	216	104
											64	286	132	-104	-48
												6084	1716	-1716	-484
													624	-484	-176
														624	176
															64

$$[\bar{A}_e] = \frac{1}{1575}$$

-1170	468	-330	132	1170	-468	330	-132	405	-162	-195	78	-405	162	195	-78
-468	0	-132	0	468	-156	132	-44	162	-54	-78	26	-162	0	78	0
-330	132	-120	48	330	-132	120	-48	195	-78	-90	36	-195	78	90	-36
-132	0	-48	0	132	-44	48	-16	78	-26	-36	12	-78	0	36	0
-1170	-468	-330	-132	1170	468	330	132	405	162	-195	-78	-405	-162	195	78
468	156	132	44	-468	0	-132	0	-162	0	78	0	162	54	-78	-26
-330	-132	-120	-48	330	132	120	48	195	78	-90	-36	-195	-78	90	36
132	44	48	16	-132	0	-48	0	-78	0	36	0	78	26	-36	-12
-405	-162	-195	-78	405	162	195	78	1170	468	-330	-132	-1170	-468	330	132
162	54	78	26	-162	0	-78	0	-468	0	132	0	468	156	-132	-44
195	78	90	36	-195	-78	-90	-36	-330	-132	120	48	330	132	-120	-48
-78	-26	-36	-12	78	0	36	0	132	0	-48	0	-132	-44	48	16
-405	162	-195	78	405	-162	195	-78	1170	-468	-330	132	-1170	468	330	-132
-162	0	-78	0	162	-54	78	-26	468	-156	-132	44	-468	0	132	0
195	-78	90	-36	-195	78	-90	36	-330	132	120	-48	330	-132	-120	48
78	0	36	0	-78	26	-36	12	-132	44	48	-16	132	0	-48	0

$$[\bar{N}_e] = \frac{1}{25200}$$

-11232	-20592	-3168	-5808	11232	-1872	3168	-528	3888	-648	-1872	312	-3888	-7128	1872	3432
-1872	-4992	-528	-1408	1872	1248	528	352	648	432	-312	-208	-648	-1728	312	832
-2958	-5598	-1047	-2007	2958	-318	1047	-87	1662	-102	-759	39	-1662	-3222	759	1479
-318	-1198	-87	-407	318	562	87	233	102	418	-39	-201	-102	-622	39	279
11232	1872	3168	528	-11232	20592	-3168	5808	-3888	7128	1872	-3432	3888	648	-1872	-312
-1872	1248	-528	352	1872	-4992	528	-1408	648	-1728	-312	832	-648	432	312	-208
2958	318	1047	87	-2958	5598	-1047	2007	-1662	3222	759	-1479	1662	102	-759	-39
-318	562	-87	233	318	-1198	87	-407	102	-622	-39	279	-102	418	39	-201
3888	648	1872	312	-3888	7128	-1872	3432	-11232	20592	3168	-5808	11232	1872	-3168	-528
-648	432	-312	208	648	-1728	312	-832	1872	-4992	-528	1408	-1872	1248	528	-352
-1662	-102	-759	-39	1662	-3222	759	-1479	2958	-5598	-1047	2007	-2958	-318	1047	87
102	-418	39	-201	-102	622	-39	279	-318	1198	87	-407	318	-562	-87	233
-3888	-7128	-1872	-3432	3888	-648	1872	312	11232	-1872	-3168	528	-11232	-20592	3168	5808
-648	-1728	-312	-832	648	432	312	208	1872	1248	-528	-352	-1872	-4992	528	1408
1662	3222	759	1479	-1662	102	-759	39	-2958	318	1047	-87	2958	5598	1047	-2007
102	622	39	279	-102	-418	-39	-201	-318	-562	87	233	318	1198	-87	-407